| Interference Press Fit Cylinder Design Calculator |  |  |  |
| :---: | :---: | :---: | :---: |
| Blocks shown as light blue are editable |  |  |  |
| Outer hub diameter $\mathrm{d}_{0}=$ | 2.0000 | in $\quad \checkmark$ |  |
| Outer hub inner diameter $\mathrm{d}_{\mathrm{h}}=$ | 1.4000 | in |  |
| shaft outer diameter $\mathrm{d}_{\mathrm{s}}=$ | 1.5100 | in |  |
| shaft inner diameter $\mathrm{d}_{\mathrm{i}}=$ | 1.0000 | in |  |
| radius within outer cylinder material $r_{0}=$ | 0.8000 | in |  |
| radius within inner cylinder material $r_{i}=$ | 0.5500 | in |  |
| contact surface diameter, in compression $\mathrm{d}_{\mathrm{c}}=$ | 1.5050 | in |  |
| contact pressure applied $\mathrm{p}_{\mathrm{c}}=$ | 3.000 | psi |  |
| modulus of elasticity $\mathrm{E}=$ | 42,000 | psi |  |
| modulus of elasticity hub $\mathrm{E}_{\mathrm{h}}=$ | 65,000 | psi |  |
| modulus of elasticity shaft $E_{s}=$ | 65,000 | psi |  |
| modulus of elasticity cast iron hub on steel shaft $E_{0}=$ | 32,000 | psi |  |
| modulus of elasticity steel shaft on cast iron hub $\mathrm{E}_{\mathrm{c}}=$ | 36,000 | psi |  |
| Poisson's ratio $\mathrm{v}=$ | 0.350 | - |  |
| Poisson's ratio $\mathrm{v}_{\mathrm{s}}=$ | 0.350 | - |  |
| Poisson's ratio $\mathrm{v}_{\mathrm{h}}=$ | 0.350 | - |  |
| Calculated Results |  |  |  |
| change in diameter of the inner member $\Delta \mathrm{d}_{\mathrm{i}}=$ | -0.00024 | in | Eq. 2 |
| change in diameter of the outer member $\Delta \mathrm{d}_{0}=$ | 0.00043 | in | Eq. 3 |
| original difference in diameters $\delta=$ | 0.00019 | in | Eq. 4 |
| $\Delta \mathrm{d}_{\mathrm{s}}=$ | 0.00015 | in | Eq. 5a |
| $\Delta d_{h}=$ | 0.00025 | in | Eq. 5b |
| (exact) total change dia. of hub and hollow shaft $\delta=$ | 0.00040 | in | Eq. 5 |
| (approx) total change dia. of hub and hollow shaft $\delta=$ | 0.00043 | in | Eq. 5c |
| shrinkage stress in the band $\sigma_{\theta}=$ | 5.19006 | psi | Eq. 6 |
| Calculated contact pressure both materials same $\mathrm{P}_{\mathrm{C}}=$ | 0.838 | psi | Eq. 7 |
| tangential stress at radius $r_{0}$ of outer cylinder $\sigma_{\theta-\mathrm{o}}=$ | 10.03610 | psi | Eq. 8 |
| tangential stress at radius $r_{i}$ of inner cylinder $\sigma_{\theta-i}=$ | -7.15332 | psi | Eq. 9 |
| radial stress at radius $r_{0}$ of outer cylinder $\sigma_{r-0}=$ | -2.20305 | psi | Eq. 10 |
| radial stress at radius ri of inner cylinder $\sigma_{r-1}=$ | 0.93224 | psi | Eq. 11 |
| tangential stress outside dia. of outer cylinder $\sigma_{\theta-00}=$ | 7.83305 | psi | Eq. 12 |
| tangential stress inside dia. of outer cylinder $\sigma_{\theta-\text { oi }}=$ | 10.83305 | psi | Eq. 13 |
| tangential stress outside dia. of inner cylinder $\sigma_{\theta-\mathrm{io}}=$ | -7.74299 | psi | Eq. 14 |
| tangential stress inside dia. of inner cylinder $\sigma_{\theta-i i}=$ | -10.74299 | psi | Eq. 15 |
| radial stress outside dia. of outer cylinder $\sigma_{r}=$ | 0.00000 | psi | Eq. 16 |
| radial stress inside dia. of outer cylinder $\sigma_{r \text {-oi }}=$ | -3.00000 | psi | Eq. 17 |
| radial stress outside dia. of inner cylinder $\sigma_{r-\text { oi }}=$ | -3.00000 | psi | Eq. 18 |
| radial stress inside dia. of inner cylinder $\sigma_{\text {r-ii }}=$ | 0.00000 | psi | Eq. 19 |
| tangential stress cast-iron hub on steel shaft $\sigma_{\theta}=$ | 3.33405 | psi | Eq. 20 |
| allowable stress for brittle materials $\sigma_{\text {all }}=$ | 1.58133 | psi | Eq. 21 |

